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(54) **FRAME ASSEMBLY FOR SNOWMOBILE**

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(57) **ABSTRACT**

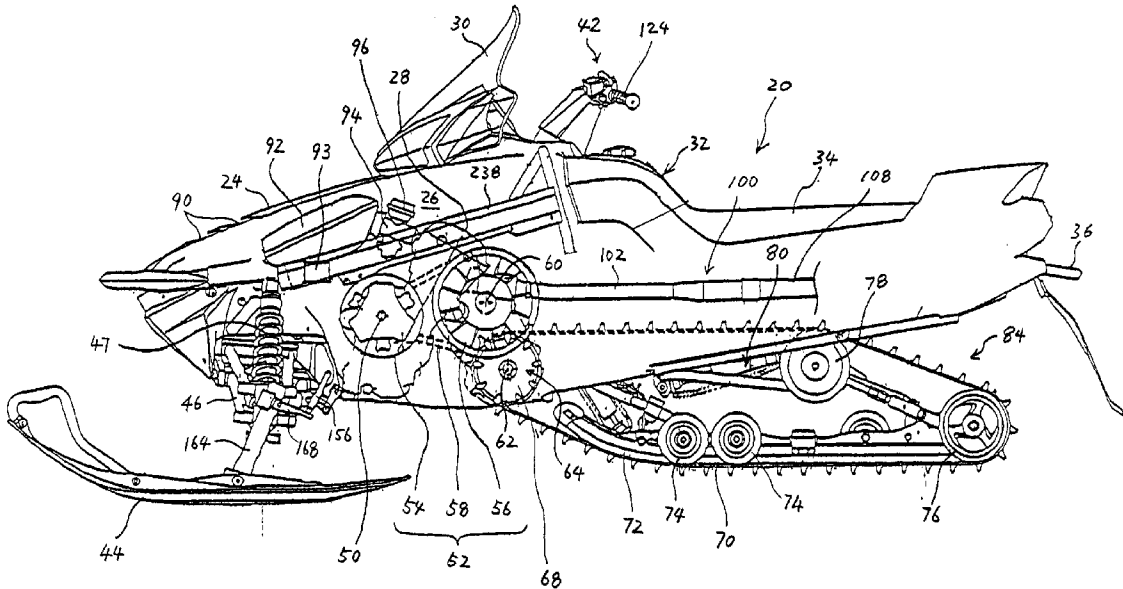
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A snowmobile has a body frame. A rear frame is coupled with the body frame. The rear frame defines a generally horizontally extending bottom section. Side sections generally vertically extend from side portions of the bottom section. A rear section generally vertically extends from a rear portion of the bottom section. The bottom, side and rear sections are unitarily formed by a casting process or a press working.



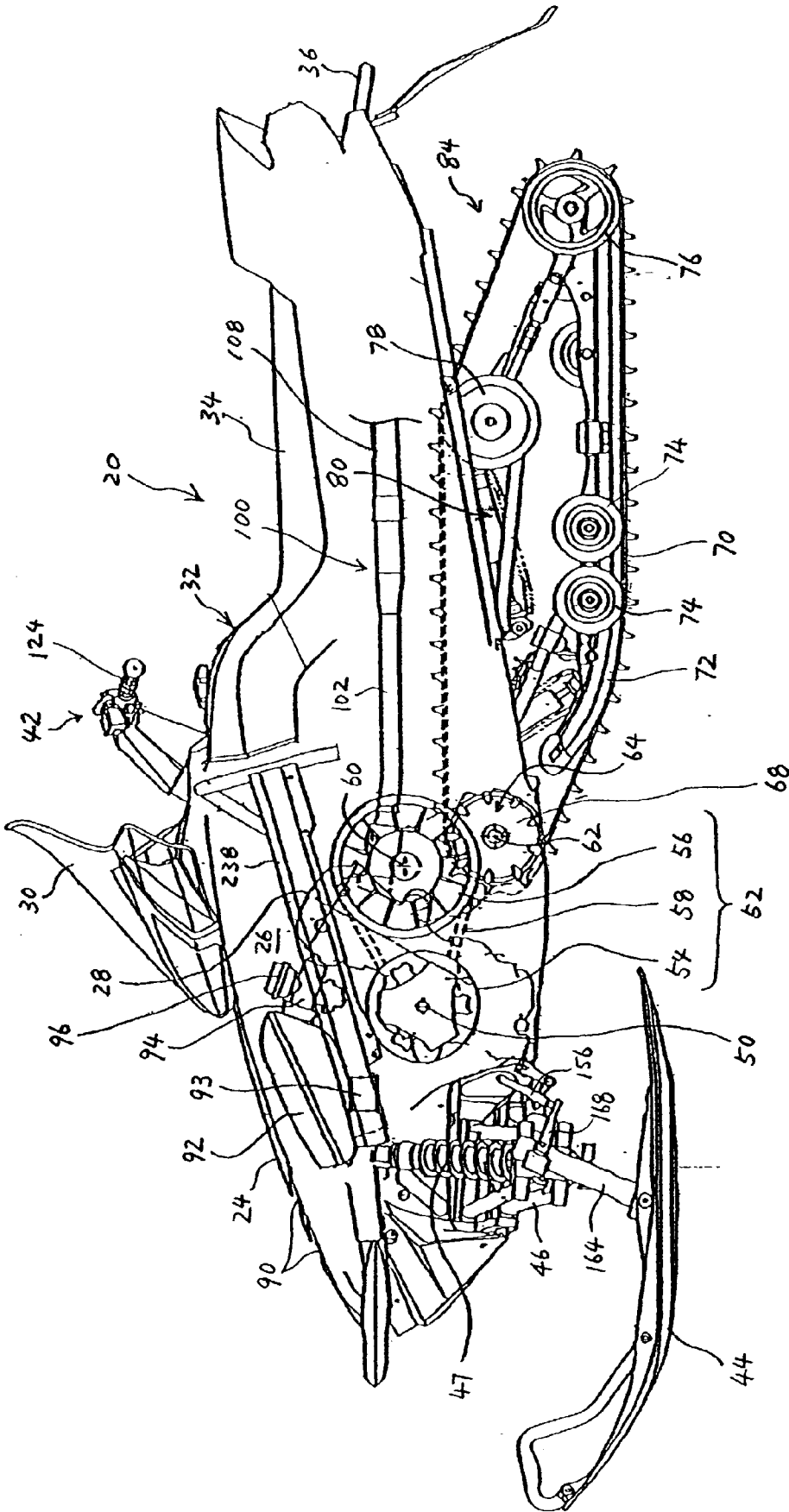


FIGURE 1

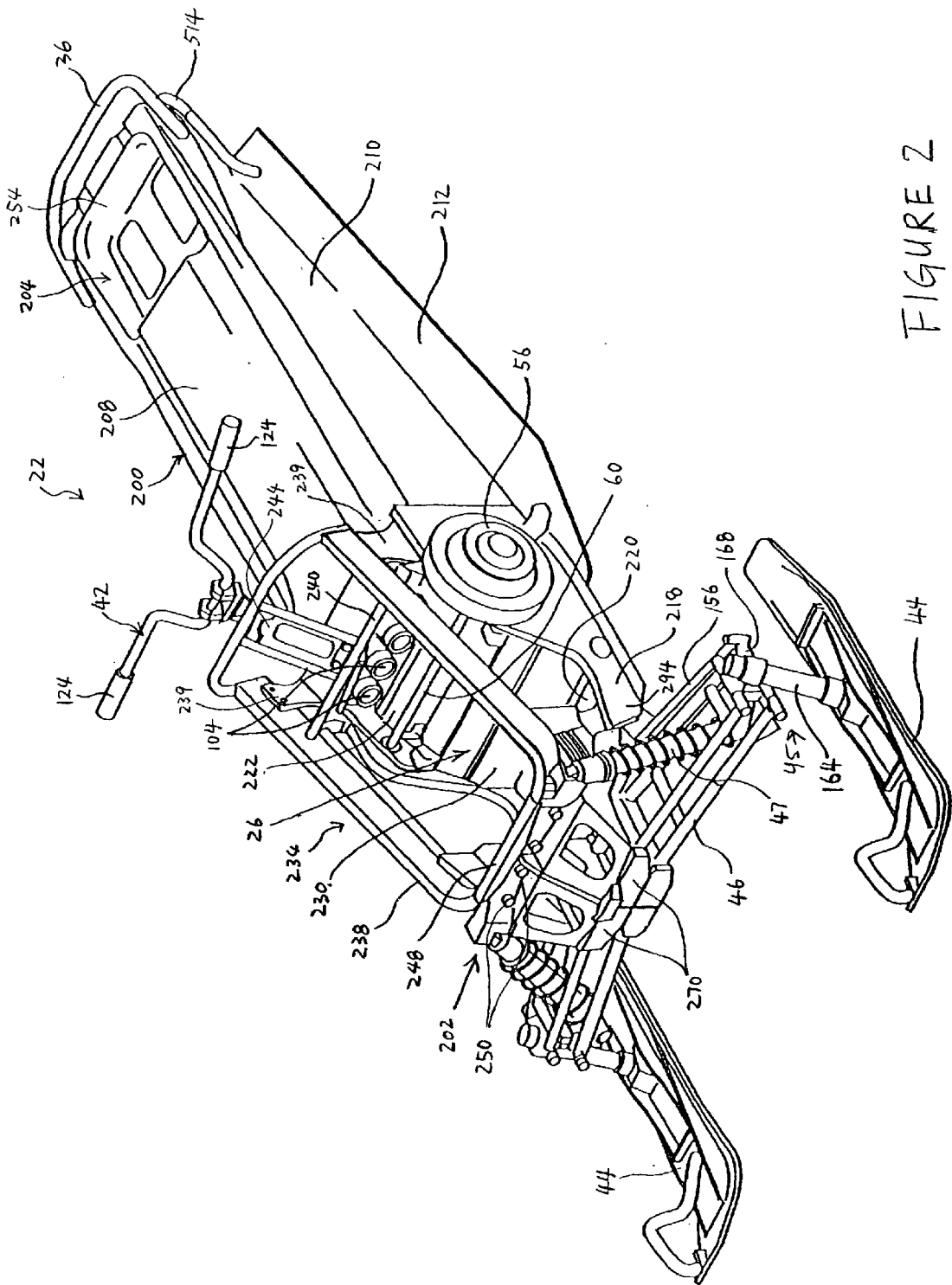


FIGURE 2

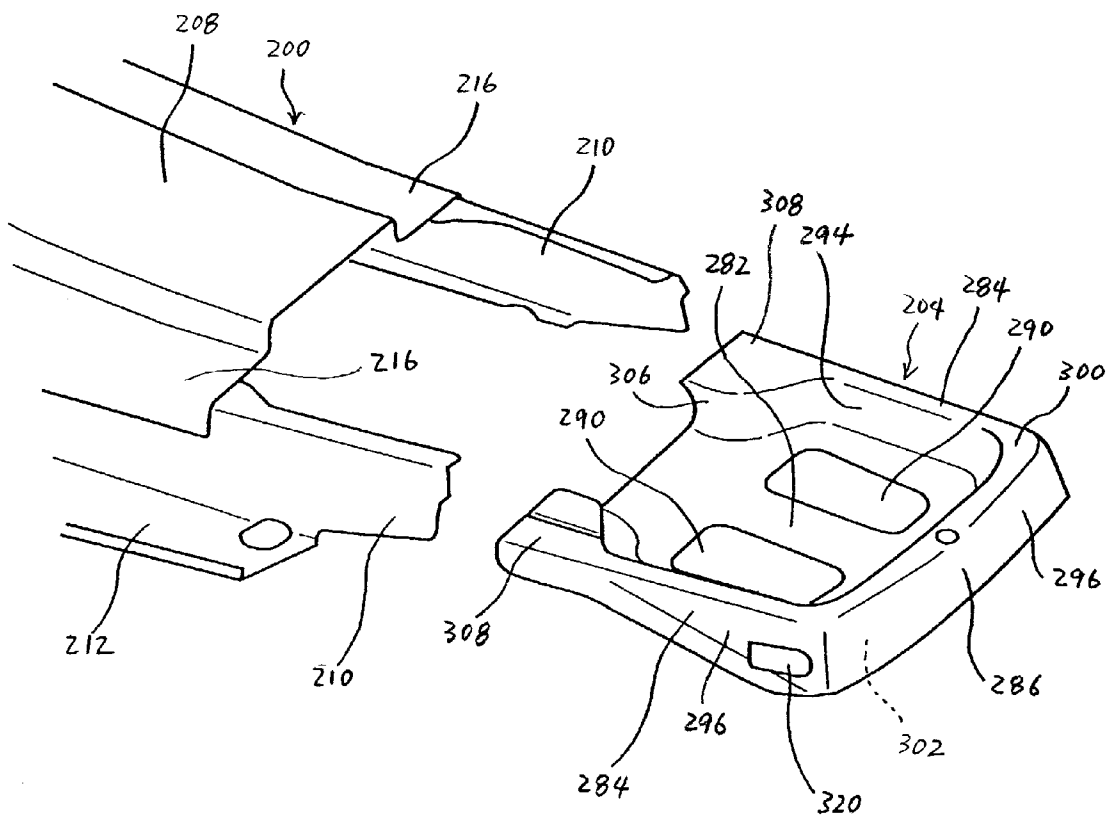


FIGURE 3

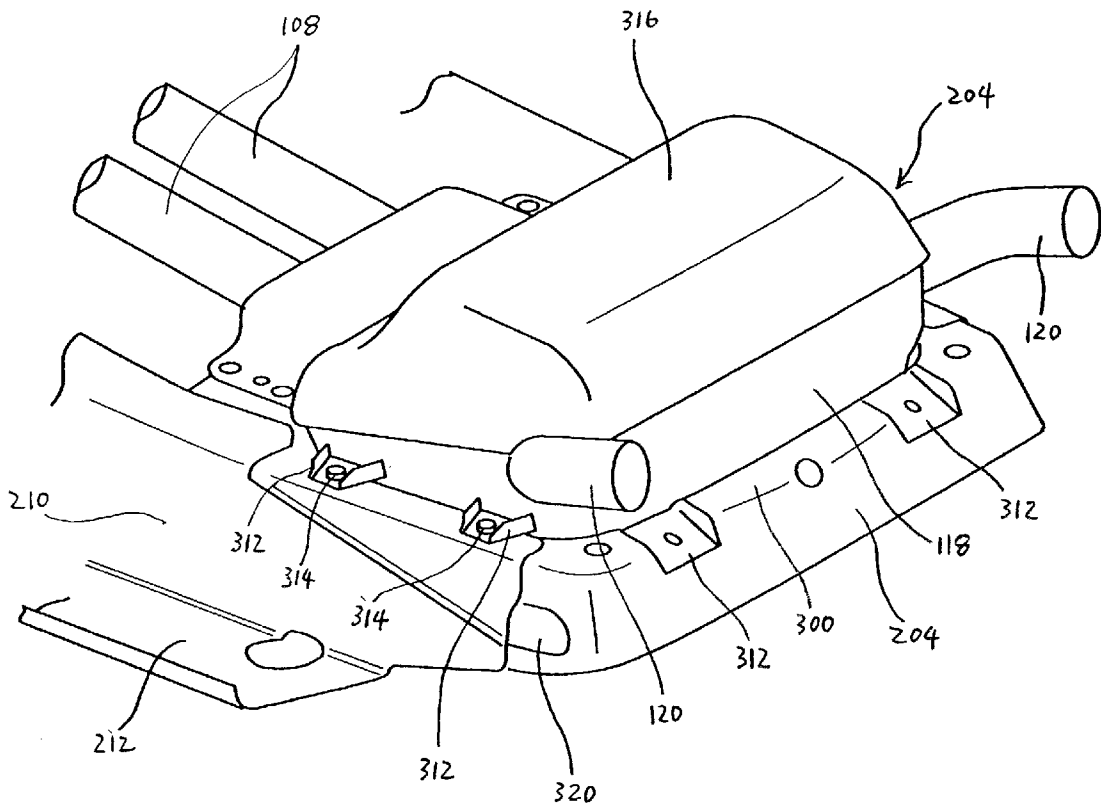


FIGURE 4

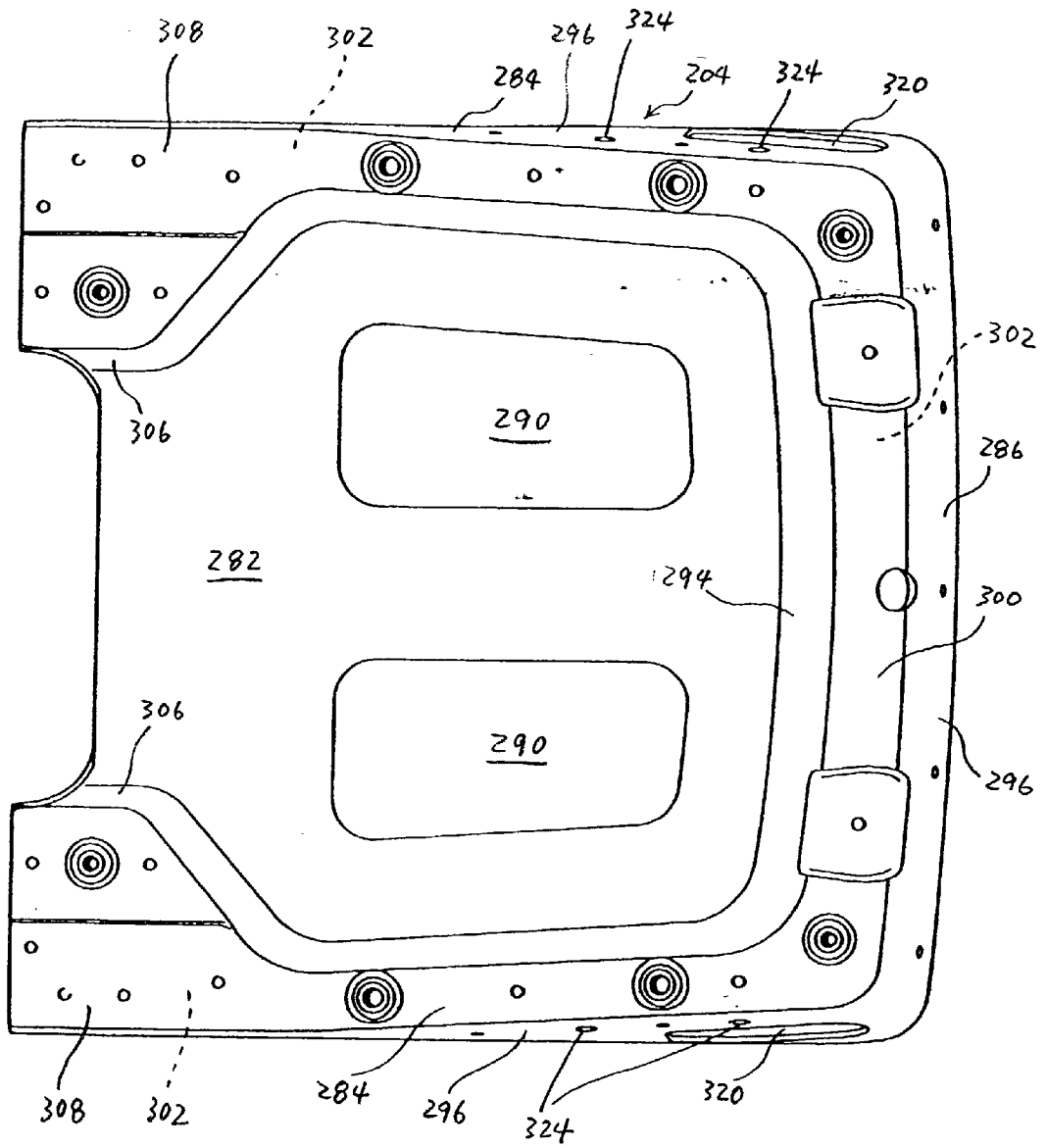


FIGURE 5

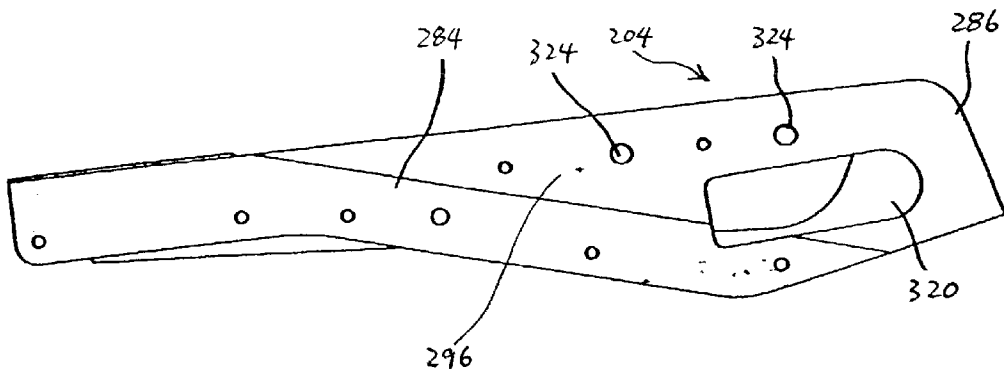


FIGURE 6

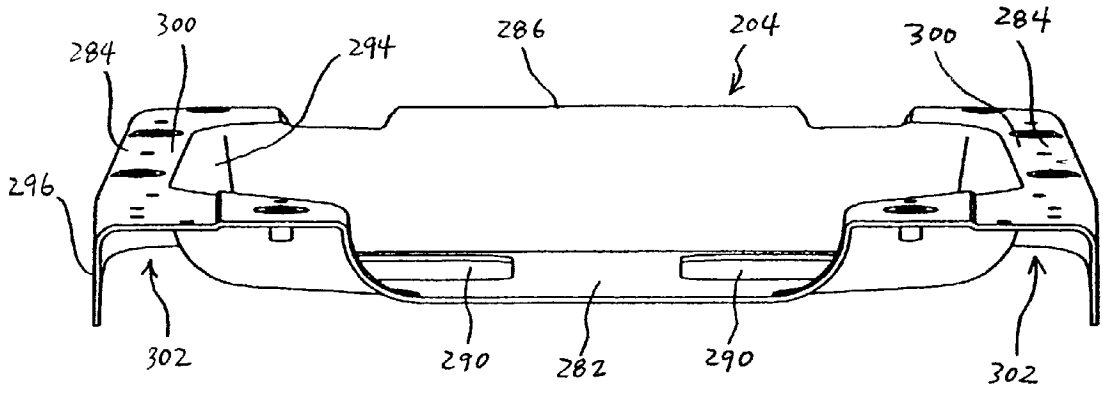


FIGURE 7

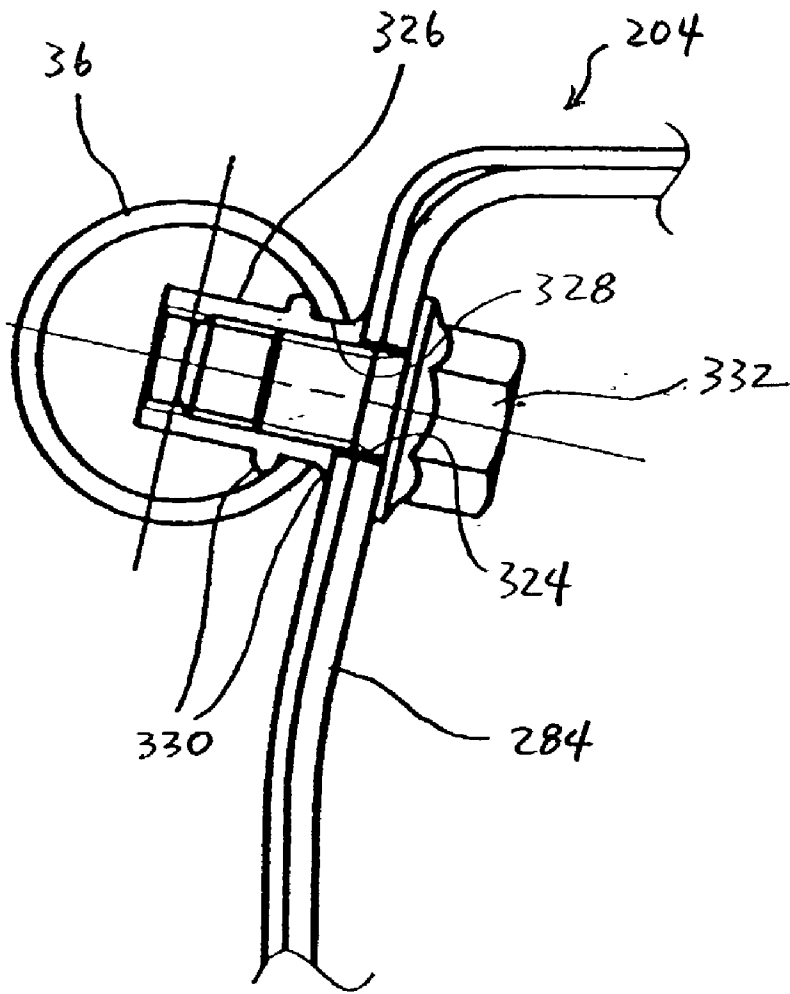


FIGURE 8

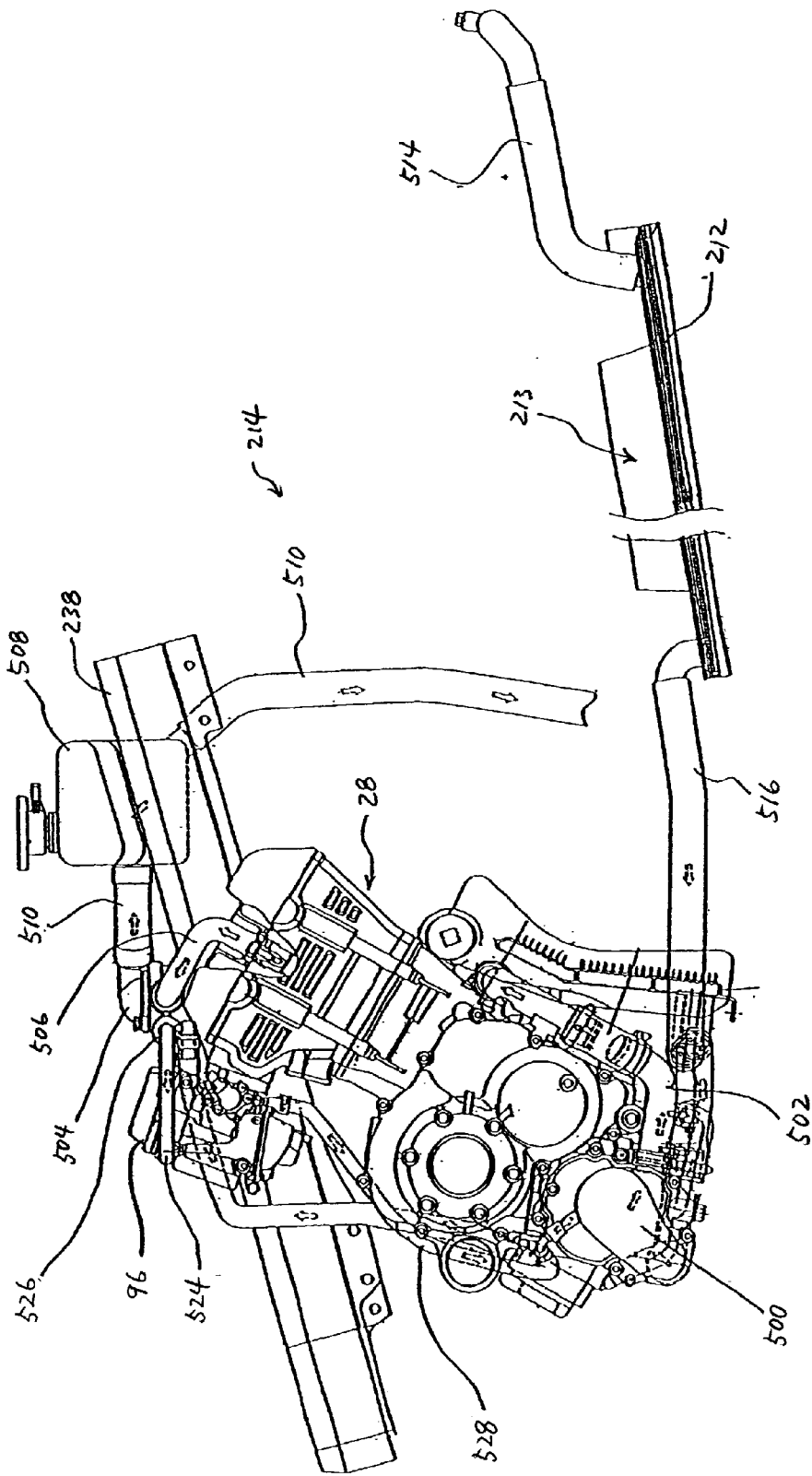


FIGURE 9

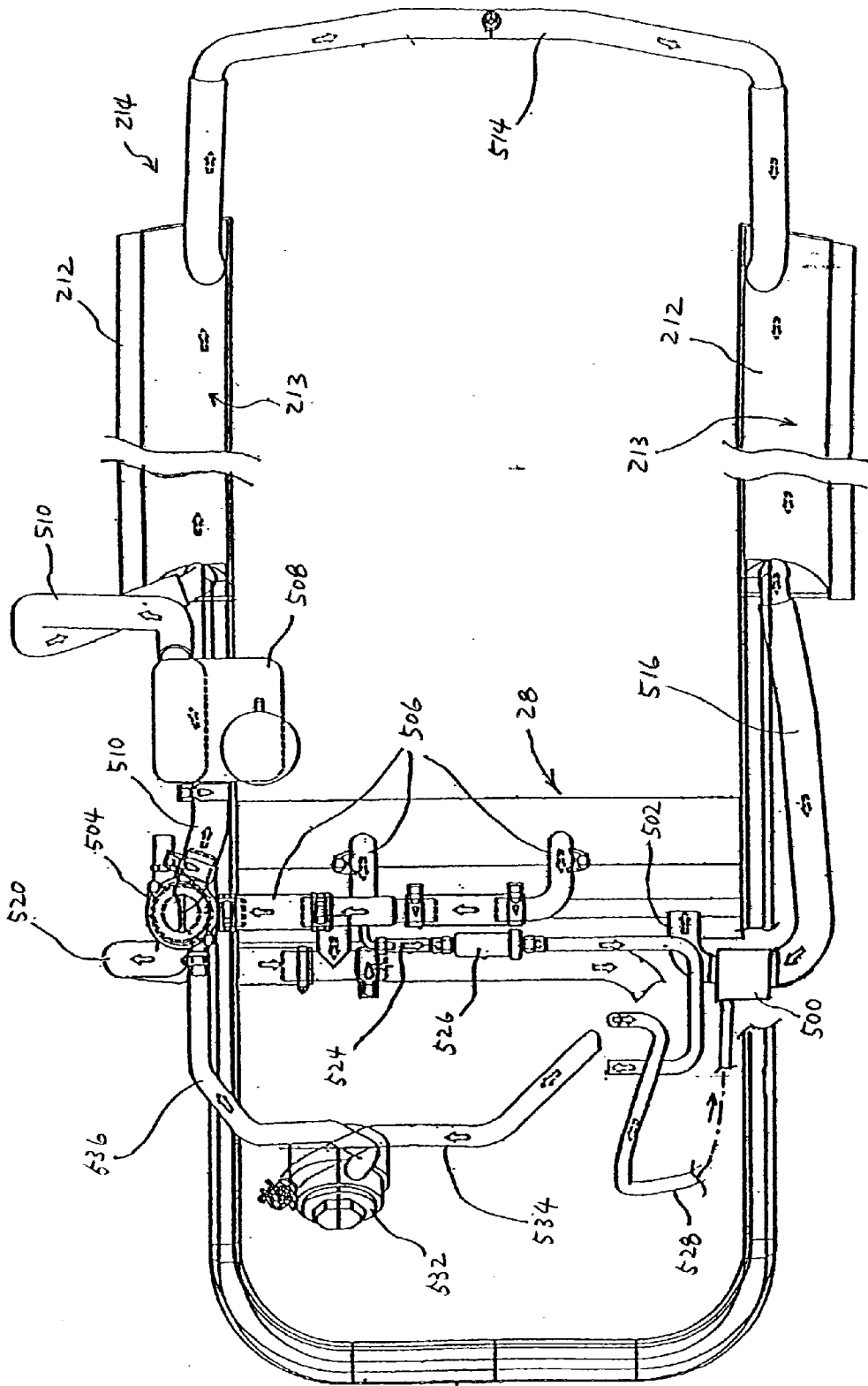


FIGURE 10

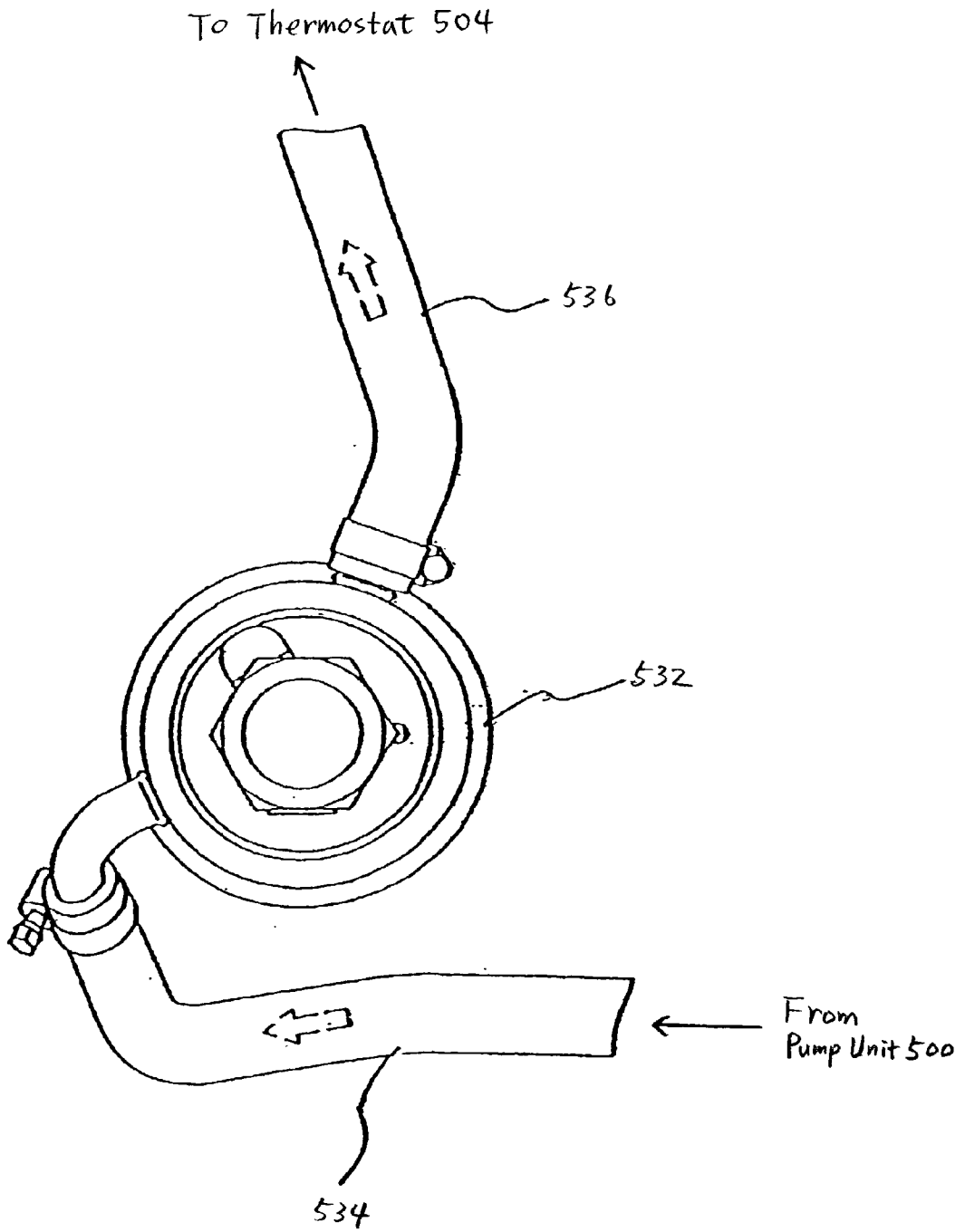


FIGURE 11

FRAME ASSEMBLY FOR SNOWMOBILE

RELATED APPLICATIONS

[0001] This application is based upon and claims priority to Japanese Patent Application No. 2001-241585, filed on Aug. 9, 2001, which is hereby incorporated by reference in its entirety.

BACKGROUND OF THE INVENTION

[0002] 1. Field of the Invention

[0003] The present invention is directed to a frame assembly for a snowmobile, and more particularly to an improved rear frame assembly for a snowmobile.

[0004] 2. Description of the Related Art

[0005] A typical snowmobile includes a frame assembly, a drive assembly including an endless drive belt for propelling the snowmobile, and a slide rail unit that supports the drive belt. The frame assembly carries an engine which powers the drive belt through a suitable transmission and drive mechanism.

[0006] Typically, a pair of front skis depends from the frame assembly for steering the snowmobile. A steering handle extends to the rider's area from the frame assembly so that the rider can operate the steering handle. A steering linkage couples the skis with the steering handle. The frame assembly also includes a rear portion, which is formed of a plurality of connected pieces extending over the drive belt and supporting the operator and passengers.

SUMMARY OF THE INVENTION

[0007] One aspect of the present invention includes the realization that the rear portion of a snowmobile frame can become deformed through use. For example, but without limitation, conventional snowmobile frames include rear portions constructed of a plurality of separate pieces that are connected with weld, rivets, bolts, or other fasteners. This portion of the frame can also be used to support a portion of the seat, a grab handle, and/or an exhaust silencer. However, after prolonged use, the rear portion of such a frame portion can become deformed. For example, the repeated heating and cooling of the exhaust silencer subjects the rear portion of the frame to thermal fatigue. Such thermal fatigue can damage and/or loosen the connections between the plurality of pieces forming the rear portion of the frame be damaged. Further, the thermal fatigue can cause some of the pieces to deform from their original shape. Additionally, loads transferred to the rear portion of the frame from the seat and/or the grab handle can damage or loosen the connections as well as deform the component pieces of the rear portion of the frame.

[0008] Deformation of the rear portion of the frame can make it more difficult to move the snowmobile by the grab handle. Additionally, damage to or loosening of the connections between the component pieces of the rear portion of the frame can cause additional noises that are annoying to the operator and passengers of the snowmobile. Where the rear portion of the frame is used to support an exhaust component, additional noises can be generated by the vibrations carried by the exhaust gasses flowing through such an exhaust component.

[0009] In accordance with one aspect of the present invention, a snowmobile comprises a frame assembly. A drive assembly depends from the frame assembly and is adapted to abut terrain so as to propel the frame assembly along the terrain. The frame assembly comprises a body frame. A rear frame is coupled with the body frame. The rear frame defines a generally horizontally extending bottom section. Side sections generally vertically extend from side portions of the bottom section. A rear section generally vertically extends from a rear portion of the bottom section. The bottom, side and rear sections being monolithically formed with each other.

[0010] In accordance with another aspect of the present invention, a snowmobile comprises a frame assembly. A drive assembly depends from the frame assembly and is adapted to abut the terrain so as to propel the frame assembly along the terrain. The frame assembly comprises a body frame. A rear frame is coupled with the body frame. The rear frame defines a generally horizontally extending bottom section. Side sections generally vertically extend from side portions of the bottom section. A rear section extends generally vertically from a rear portion of the bottom section. Each one of the side and rear sections forms an inner wall, an outer wall and a top wall connecting the inner and outer walls. The inner and outer walls are spaced apart from each other.

BRIEF DESCRIPTION OF THE DRAWINGS

[0011] These and other features, aspects and advantages of the present invention will be better understood with reference to a preferred embodiment, which is illustrated in the accompanying drawings. The illustrated embodiment is merely exemplary and is not intended to define the outer limits of the scope of the present invention. The drawings of the illustrated arrangement comprise eleven figures.

[0012] **FIG. 1** is a side elevational view of a snowmobile having a frame assembly arranged and configured in accordance with certain features, aspects and advantages of the present invention. Certain portions are illustrated as being broken away and other internal portions are shown in hidden line.

[0013] **FIG. 2** is a perspective view of the frame assembly of **FIG. 1** with certain components, such as an engine, being removed.

[0014] **FIG. 3** is a partial perspective and exploded view of a rear portion of the frame assembly.

[0015] **FIG. 4** is a partial perspective view of the frame assembly shown in **FIG. 3** with the rear portion of the frame assembly assembled and an supporting an exhaust component.

[0016] **FIG. 5** is a top plan view of the rear portion of the frame illustrated in **FIG. 3**.

[0017] **FIG. 6** is a side elevational view of the rear portion shown in **FIG. 5**.

[0018] **FIG. 7** is a front elevational view of the rear portion shown in **FIG. 5**.

[0019] **FIG. 8** is an enlarged and partial cross-sectional view of a grab handle attached to the rear portion shown in **FIG. 5**.

[0020] FIG. 9 is a schematic side elevational view of an engine cooling system of the snowmobile shown in FIG. 1. A portion of the frame assembly is illustrated in the figure.

[0021] FIG. 10 is a schematic top plan view of the engine cooling system shown in FIG. 9.

[0022] FIG. 11 is an enlarged and partial front elevational view of the engine cooling system shown in FIG. 9 and showing a portion thereof around an oil cooler of the snowmobile.

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENT

[0023] With particular reference to FIGS. 1, 2 and 4, an overall construction of a snowmobile 20 including certain features, aspects and advantages of the present invention is described below.

[0024] The snowmobile 20 generally comprises a frame assembly 22 (FIG. 2) that carries a number of other components of the snowmobile 20. The frame assembly 22 is described below in greater detail. A forward body cover 24 is disposed over a forward portion of the frame assembly 22. As illustrated in FIG. 1, the forward body cover 24 covers, in part, an engine compartment 26 in which an engine 28 is mounted. The engine 28 preferably is an in-line, four-cylinder, water-cooled, four-cycle engine and is described in greater detail below.

[0025] A windshield 30 is disposed over a mid-portion of the body cover 24. The windshield 30 provides some degree of protection for the riders from wind and other elements during operation of the snowmobile 20. Rearward of the windshield 30, a fuel tank 32 is mounted to the frame assembly 22 in a manner that allows the body cover 24 and the fuel tank 32 to blend together for aesthetic and other reasons.

[0026] Rearward of the fuel tank 32, a seat 34 is mounted to the frame assembly 22. Rearward of the seat 34 is positioned a grab bar 36 that can be used to raise a rear portion of the snowmobile 20 for turning and maneuvering when the snowmobile 20 is not being ridden.

[0027] Forward of the seat 34 and the fuel tank 32 is a steering handle assembly 42. The steering handle assembly 42 can carry appropriate controls and is coupled to a pair of front skis 44 by a steering linkage (not shown). Each front ski 44 has a pivot shaft which is supported by a holder 45. The holders 45 are mounted to the frame assembly 22 through a pair of swing units 46. As the handle assembly 42 is turned, the pivot shafts of the skis 44 pivot clockwise and counterclockwise about axes of the holders 45. As the skis 44 pivot, the direction of the snowmobile 20 can be altered.

[0028] A pair of suspension assemblies 47 also is provided between the frame assembly 22 and the skis 44. Each suspension assembly 47 preferably comprises a shock absorber unit (for example, a cylinder type) and a coil spring to absorb shocks from the rough terrain and vibrations made by the rough terrain, although any other construction can be applied.

[0029] The engine 28 is mounted to the frame assembly 22 in any suitable manner. A set of resilient engine mounts can be used to secure the engine 28 to the frame assembly 22. By mounting the engine 28 with the resilient engine mounts,

vibrations caused by operation of the engine 28 are reduced or eliminated before transmission through the frame assembly 22 to the operator.

[0030] With reference to FIG. 1, the engine 28 in the illustrated arrangement is inclined rearwardly relative to vertical, and is mounted transversely within the engine compartment 26. Cylinders of the engine 28 extend side-by-side across a width of the snowmobile 20. In some arrangements, engines having differing numbers of cylinders, different cylinder configurations (e.g., V, W, opposing, etc.), different orientations (e.g., vertical) and different operating principles (e.g., two-stroke, rotary, diesel, etc.) can be used.

[0031] The engine 28 also comprises a crankshaft (not shown) that extends transversely and an output shaft 50 that extends parallel to the crankshaft and is driven by the crankshaft. The output shaft 50 drives a transmission, which is a continuously variable transmission 52 in the illustrated arrangement. Other transmissions also can be used. In the illustrated arrangement, the transmission 52 comprises a drive pulley 54, a driven pulley 56 and a v-belt 58. The output shaft 50 rotates the drive pulley 54. The output shaft 50 and the drive pulley 54 can be connected together through a clutch, a centrifugal clutch, a sprag clutch or can be directly connected together.

[0032] The drive pulley 54 powers the driven pulley 56 with the v-belt 58 in the illustrated arrangement. In some configurations, a drive chain can be used in place of the v-belt 58. Other arrangements also can be used. The driven pulley 56 is connected to and rotates about a transfer shaft 60. In the illustrated arrangement, the transfer shaft 60 carries a sprocket (not shown) at the end opposite to the driven pulley 56. The sprocket is connected to a further sprocket that is carried by a drive shaft 62. The sprockets are connected together by a chain in one arrangement and these three components are disposed within a chain box (not shown).

[0033] The drive shaft 62 powers a drive unit 64. The drive unit 64 generally comprises a plurality of drive wheels 68. The illustrated drive unit 64 comprises four drive wheels 68. The drive wheels 68 provide a motive force to a drive belt 70, which is commonly used in the snowmobile industry. The drive belt 70 abuts the terrain.

[0034] The drive belt 70 is guided around a preferred path on a pair of slide rails 72, a plurality of suspension wheels 74 and main rear suspension wheels 76. The slide rails 72 preferably support the suspension wheels 74 and the main rear suspension wheels 76. An idler roller 78 preferably is mounted to the frame assembly 22 and helps to define the preferred path for the drive belt 70. As is known in the snowmobile industry, these components can be mounted to the frame assembly 22 with a rear suspension system 80. Any suitable rear suspension system 80 can be used and certain portions of the rear suspension system 80 have been schematically illustrated in the illustrated arrangement.

[0035] The drive belt 70, the slide rails 72, the suspension wheels 74, the main rear suspension wheels 76, the idle roller 78 and the rear suspension system 80 generally form a drive assembly 84 in the illustrated embodiment.

[0036] With continued reference to FIG. 1, air is drawn into the engine compartment 26 through suitable air pas-

sages. In some arrangements, the air is drawn through ventilation openings **90** formed in the body cover **24**. The air drawn or forced into the engine compartment **26** circulates about the engine **28** and related drive components to help cool the engine **28** and the related drive components.

[**0037**] The air also is drawn into an air intake box **92**. The air intake box **92** is disposed forward of the engine **28** in the illustrated arrangement. The air intake box **92** can be mounted to the frame assembly **22** in a suitable manner. An inlet **93** into the air intake box **92** can extend upward into a lower surface of the air intake box **92**.

[**0038**] A set of intake runners **94** extends between the illustrated air intake box **92** and the engine **28**. Preferably, a charge former **96** is disposed along each of the intake runners **94**. Advantageously, the intake runners **94** extend directly rearward to the engine **28**.

[**0039**] One of the charge formers **96** preferably correspond to each cylinder. In some arrangements, a single charge former can be used upstream of a separation point for runners extending to individual cylinders. In addition, in the illustrated arrangement, the engine **28** is carbureted. Alternatively, the charge formers **96** can be fuel injectors that are mounted for direct injection, indirect injection or port injection.

[**0040**] The air-fuel charge provided in this manner is combusted within the engine in a suitable manner. The combustion byproducts then are exhausted through a suitable exhaust system **100**. In the illustrated arrangement, the exhaust system **100** extends directly rearward from the engine **28**. In this manner, an exhaust runner **102** that extends rearward from the engine **28** can be tuned to the engine **28** for improved engine performance. Additionally, the length of each runner **102** can be lengthened prior to merging together with any other runners such that pulse effects on adjoining cylinders can be reduced. The exhaust runners **102** correspond to each cylinder and extend generally rearward from the engine **28**. Each exhaust runner **102** is coupled to the engine **28** through openings **104** (**FIG. 2**) in the illustrated arrangement. In some arrangements, a single manifold can be used.

[**0041**] With continued reference to **FIG. 1**, the runners **102** preferably are joined to merge into manifold pipes **108**. The manifold pipes **116** extend rearward to a silencer box **118** (**FIG. 4**). The silencer box **118** provides an enlarged volume into which the exhaust can flow. Exhaust energy is dissipated within the silencer box **118** and the noise level of the exhaust can be decreased. The silencer box **118** preferably is disposed rearward of the seat **34**. A preferred position and arrangement of the silencer box **118** is described in greater detail below.

[**0042**] A pair of exhaust pipes **120** (**FIG. 4**) preferably extends rearward from the silencer box **118**. In some arrangements, a single exhaust pipe can extend from the silencer box **118**. One end of the exhaust pipes **120** preferably defines an ultimate exhaust discharge from the snowmobile **20** such that the exhaust gases are discharged into the atmosphere.

[**0043**] With reference to **FIGS. 2 and 3**, a general construction of the frame assembly **22** is described in greater detail below.

[**0044**] The frame assembly **22** generally comprises a body frame **200**, a front frame **202** and a rear frame **204**. The body frame **200** is disposed center of the frame assembly **22** and forms a main body of the snowmobile **20**. The front frame **202** is disposed forward of the body frame **200** and forms a front body of the frame assembly **22**. The rear frame **204** is disposed rear of the body frame **200** and forms a rear body of the frame assembly **22**.

[**0045**] The body frame **200** preferably comprises an upper horizontal section **208**, side vertical sections **210** and lower horizontal sections **212**. The upper horizontal section **208** extends fore to aft and forms a center area of the body frame **200**. The side vertical sections **210** extend vertically downwardly from both sides of the upper horizontal section **208**. The lower horizontal sections **212**, in turn, extend horizontally outwardly from lower ends of the side vertical sections **210**.

[**0046**] The lower horizontal sections **212** preferably form foot step areas. In the illustrated arrangement, heat exchange devices **213** (see **FIGS. 9 and 10**) are unitarily formed with the respective lower horizontal sections **212**. The heat exchange devices **213** and an engine cooling system **214** (see **FIGS. 9 and 10**) that includes the heat exchange devices **213** are described in greater detail below with reference to **FIGS. 9 and 10**.

[**0047**] Preferably, as shown in **FIG. 3**, the upper horizontal section **208** is formed separately from the other part of the body frame **200**. The upper horizontal section **208** can be made of, for example, a sheet metal. The upper horizontal section **208** also includes side portions **216** which are positioned lower than a center portion thereof to reinforce the section **208**.

[**0048**] The upper horizontal section **208** can be formed by, for example, a press working. The side vertical sections **210** and the lower horizontal sections **212** preferably are unitarily formed with each other and are made of, for example, pressed sheet metal, or are made of an extruded aluminum alloy material. The side vertical sections **210** extend beyond rear ends of the upper and lower horizontal sections **210**, **212**. Each side vertical section **210** has a flapped top portion which has a smaller area than the side **216** of the upper horizontal section **208**.

[**0049**] The sections **208**, **210**, **212** can be entirely unitarily formed with each other. To the contrary, the sections **208**, **210**, **212** can be formed from separate pieces, respectively. Additionally, the sections **208**, **210**, **212** can be produced by any methods other than the methods described above. For example, the sections **208**, **210**, **212**, unitarily formed or separately formed, can be produced by a mold casting, die-casting or lost-wax casting process. Similarly, the other components and/or members described below also can be formed with any metal or plastic material and can be formed by any conventional method regardless of being expressly described or not, unless being otherwise explained.

[**0050**] The illustrated body frame **200** additionally comprises side panels **218**, a floor panel **220** and a frame member **222**, which preferably are produced by a mold casting, die-casting or lost-wax casting process.

[**0051**] The side panels **218** are affixed to the respective side sections **210** by welding or by fasteners such as, for example, bolts or rivets. The side panels **218** define openings

(not shown) through which the transfer shaft **60** (FIG. 1) extends. The driven pulley **56** is positioned aside the side panel **218** on the left side. The floor panel **220** extends between lower ends of the side panels **218** to form a bottom portion of the engine compartment **26**.

[0052] The frame member **222** extends between upper portions of the side panels **218** and generally covers a front end of the upper horizontal section **208**. The frame member **222** defines the rear engine mounts **48** and the openings **104** for the exhaust runners **102** (FIG. 1).

[0053] A further heat exchange device **230** for the engine **28** is disposed below the frame member **222**. Cooling water for cooling the engine **28** also circulates through the heat exchange device **230**. The heat exchange device **230** preferably is affixed to the side panels **218** and the frame member **222** and form part of the frame assembly **22** together with the side panels **218**.

[0054] The body frame **200** thus is rigidly formed with the foregoing components and members. To further increase the rigidity of the frame assembly **22** and to be coupled with the front frame **202**, the frame assembly **22** preferably includes a frame reinforcement **234** (FIG. 2) secured to the body frame **200**. This reinforcement **234** desirably is generally rectangularly shaped and advantageously is disposed generally above the engine **28**. Thus, the engine **28** is positioned within a cavity defined between the side panels **218**, above the floor panel **220**, forward of the frame member **222** and below the reinforcement **234**.

[0055] The illustrated reinforcement **234** preferably comprises a generally rectangular tubular member **238** that is bent into a U-shape. The tubular member **238** preferably is made of an extruded aluminum alloy material. Rear ends of the tubular member **238** are affixed to upwardly extending sections **239** of the frame member **222**. A bight of the U-shape is disposed forwardly in the illustrated arrangement. A bar **240** extends between the two legs of the U-shaped tubular member **238** at the end opposite the bight. The bar **240** reinforces the open end of the U-shape defined by the tubular member **238**. A second bar **242** extends upward and between rear ends of the tubular member **238** to further reinforce the reinforcement **234**. Furthermore, a support unit **244** extends upward and rearward from the bar **240** to the second bar **242**. The steering handle assembly **42** preferably is affixed to the support unit **244**.

[0056] A holder member **248** is coupled with a front portion of the tubular member **238** by welding or by fasteners such as, for example, bolts or rivets. The holder member **248** preferably is made of sheet metal and is formed by a bending process. The front portion of the tubular member **238** is nested in a recessed portion defined by the holder member **248**. Four bolts **250**, for example, are mounted at the holder member **248** with threaded portions thereof extending generally upwardly and forwardly.

[0057] With continued reference to FIG. 2, the front frame **202** preferably comprises a pair of generally symmetrically formed front members **270** which are coupled with each other. Preferably, the front members **270** are made of aluminum alloy and are formed by, for example, a mold casting, a die-casting or lost-wax casting process. Each side portion of the front members **270** is coupled with a respective side panel **218** by welding or by suitable fasteners such as, for example, bolts or rivets.

[0058] The front member **270** supports the swing units **46** on the respective side portions. Each swing unit **46** in the illustrated arrangement is formed with a pair of linked tubular members disposed above and below each other, although any other constructions can be applied. Each side portion of the front members **270** defines a recess (not shown) recessed inwardly. Each swing unit **46** is placed at the recess and is swingably journaled on the side portion.

[0059] A top side portion of each front member **270** forms a mount boss. A top end of each suspension assembly **47** is journaled at the mount boss for pivotal movement. A lower end of each suspension assembly **47** is affixed to a portion of the lower linked member of each swing unit **46** where the holder **45** is attached, or is directly affixed to the holder **164**.

[0060] With reference to FIGS. 9-11, the engine cooling system **214** and the heat exchange devices **213** are described below in greater detail.

[0061] The engine cooling system **214** preferably is a closed-loop type and has a water pump unit **500**. The pump unit **500** is connected to an inlet of a water jacket (not shown) of the engine **28** through a delivery conduit **502** to pressurize the cooling water to the water jacket. An outlet of the water jacket is connected to an inlet of a thermostat **504** through a delivery conduit **506**. An outlet of the thermostat **504** is connected to an inlet of a water reservoir **508** through a delivery conduit **510**. An outlet of the water reservoir **508** is connected to an inlet of the heat exchange device **213** on the right hand side. As described above, the heat exchange device **213** on this side is unitarily formed with the respective lower horizontal section **212**.

[0062] An outlet of the heat exchange device **213** on the right hand side is connected to an inlet of the heat exchange device **213** on the left hand side through a delivery conduit **514**. Similarly to the right hand side construction, the heat exchange device **213** on this side is unitarily formed with the corresponding lower horizontal section **212**. An outlet of the heat exchange device **213** on the left hand side is connected to an inlet of the pump unit **500** through a delivery conduit **516**.

[0063] The cooling circuit described above is a primary route of the cooling water for the engine cooling system **214**. The cooling water is pressurized by the pump unit **500** to the water jacket of the engine **28**. The water travels through the water jacket and moves to the thermostat **504**. The thermostat **504** allows the water to move to the heat exchange device **213** on the right hand side through the water reservoir **508** if a temperature of the water exceeds a preset temperature. If the water temperature does not exceed the preset value, the thermostat **504** prevents the water from moving to the heat exchange device **213**.

[0064] The engine cooling system **214** also includes a bypass. The bypass includes a bypass conduit **520** connecting the outlet of the thermostat **504** to the inlet of the pump unit **500**. When the thermostat is closed, the water flows through the bypass and re-circulates through the engine water jacket without passing through the heat exchange devices **213**. Thus the cooling water is more quickly heated to the preset temperature.

[0065] The engine cooling system **214** also includes a further cooling circuit for adjusting a temperature of the charge former **96**, which is a carburetor in the illustrated

embodiment. A coolant delivery conduit **524** connects the conduit **506** with an inlet of the carburetor **96**. Another thermostat **526** is disposed in-line in the delivery conduit **524** to adjust a flow of the water through the charge former **96** according to the temperature of the water therein. The outlet of the water jacket in the carburetor **96** is connected to the inlet of the pump unit **500** through a delivery conduit **528**.

[0066] The engine cooling system **214** has a yet another cooling circuit for cooling the lubricating oil circulated through the engine **28**. An inlet end of a delivery conduit **534** is connected to the delivery conduit **502** at a position adjacent to the outlet of the pump unit **500**. The outlet of the delivery conduit **534** is connected to an inlet of a water jacket disposed in thermal communication with oil within an oil cooler **532**. An outlet of the oil cooler water jacket is connected to the inlet of the thermostat **504** through a delivery conduit **536**. The oil cooler **532** forms part of a lubrication system of the engine **28**. The water in the primary route thus can absorb heat from lubricant oil in the lubrication system and transfer the heat to the heat exchange devices **213**.

[0067] With reference to FIGS. 2 and 3 and with additional reference to FIGS. 5-8, the rear frame **204** is described below in greater detail.

[0068] The rear frame **204** preferably comprises a single rear member. The illustrated rear frame **204** defines a generally horizontally extending bottom section **282**. Side sections **284** extend generally vertically from the periphery of the bottom section **282**. A rear section **286** extends generally vertically from a rear portion of the bottom section **282**. Preferably, the rear frame **204** is made of a casting and is formed by, for example, a mold casting, die-casting or lost wax-casting process. Alternatively, the rear member frame **204** can be made of a sheet metal and shaped through a press working. Other materials and methods, of course, are applicable.

[0069] The bottom section **282** preferably forms a generally flat and relatively broad area. The illustrated bottom section **282** also forms a pair of openings **290** which are disposed side by side. The drive belt **70** in part extends under the openings **290**. Thus, the snow raised by the drive belt **70** while the snowmobile **20** proceeds along the terrain can enter the rear frame **204** through the openings **290** and can cool the silencer box **118** which is located on the rear frame **204** as described below.

[0070] The side and rear sections **284**, **286** preferably define an inner wall **294** and an outer wall **296**. The inner wall **294** is continuously formed through the side and rear sections **284**, **286**. The outer wall **296** also is continuously formed through the side and rear sections **284**, **286**. The inner and outer walls **294**, **298** are spaced apart from each other and a top wall **300** connects the inner and outer walls **294**, **298**. Thus, a channel **302** (FIG. 7) is formed under the top wall **300**.

[0071] Preferably, at least a portion of at least one of the inner and outer walls **294**, **296** are beveled. For example, at least a portion of the inner wall **294** preferably extends from the bottom section **282** at an angle between 0° and 90° . Additionally, at least a portion of the inner wall preferably extends from the top wall **300** at an angle between 0° and

90° . Further, at least a portion of the outer wall **296** extends from the top wall **300** at an angle between 0° and 90° .

[0072] At the forward portions **306** of the inner wall **294**, the spacing between the outer wall **298** and the inner wall **294** is greater than the spacing along the remainder of the inner wall **294**. Accordingly, each forward portion **308** of the top wall **300** corresponding to the forward portion **306** of the inner wall **294** has a broader area than the remainder of the top wall **300**. The rear frame **204** thus has a generally C-shaped peripheral rampart. This C configuration is advantageous because the rampart enhances rigidity of the rear frame **204** and also provides a sufficient recessed space.

[0073] With reference to FIG. 3, the rear portion of the body frame **200** extends slightly upwardly and rearwardly. In the illustrated embodiment, rear ends of the side vertical sections **210** extend in such a way. The forward portion of the rear frame **204** is fitted into a space formed between the rear ends of the side vertical sections **210**. Accordingly, the rear frame **204** extends slightly upwardly and rearwardly.

[0074] With reference to FIG. 4, the silencer box **118** preferably is placed in the recessed space surrounded by the rampart. The illustrated silencer box **118** is configured to nest with the recessed space and is affixed to the rear frame **204** with brackets **312** extending from the silencer box **118** affixed to the top wall **300** by bolts **314**. The silencer box **118** preferably is formed with upper and lower members. The members can be made of sheet metal and is formed by, for example, a press working. The upper and lower members are coupled with each other.

[0075] The combination of the rear frame **204** with the silencer box **118** can have sufficient rigidity and the silencer box **118** can be well protected. In addition, the silencer box **118** can have enough volume because the silencer box **118** is placed at the recessed space.

[0076] With reference to FIG. 4, an upper half of the silencer box **118** is covered with a heat isolating cover **316** via a heat insulator (not shown). The illustrated exhaust pipes **120** extend from rear and side corners of the silencer box **118**. Alternatively, the exhaust pipes **120** can extend from a bottom surface of the silencer box **118**.

[0077] A portion of the delivery conduit **514** of the engine cooling system **214** preferably extends through the channel **302** defined under the top wall **300**. Slots **320** are formed on both sides of the outer wall **296**. Each slot **320** preferably is positioned at a rear portion of the outer wall **296**. The delivery conduit **514** enters the channel **302** through one of the slots **320** and exits through the other slot **320**. The portion of the delivery conduit **514** thus is nested within a portion of the groove **302** that extends transversely across the rear frame **204**.

[0078] As such, the rear frame **204** can protect the portion of the delivery conduit **514**. In addition, because the channel **302** opens downwardly, the snow raised by the drive belt **70** during operation can splash onto the delivery conduit **514**. The water flowing through the delivery conduit **514** can thus be efficiently cooled by the snow.

[0079] With reference to FIG. 8, the grab bar **36** preferably is affixed to the side sections **284** of the rear frame **204**. Apertures **324** (FIGS. 5, 6 and 8) are formed on the outer wall **296**. Nuts **326** preferably extend into the grab bar **36**,

which preferably is a pipe, through apertures 328 formed on the grab bar 36. The nuts 326 are pulled with high force so as to plastically deformed and thus form small projections 330. The projections 330 fix the nuts 326 onto the grab bar 36. Bolts 332 are inserted into the apertures 324 of the rear frame 204 and then into the nuts 326. The grab bar 36 thus is affixed to the side section 284 of the rear frame 204.

[0080] The same snowmobile is disclosed in co-pending U.S. applications, which attorney docket numbers are FY.50084US0A and FY.50086US0A, both titled FRAME ASSEMBLY FOR SNOWMOBILE, the entire contents of which are hereby expressly incorporated by reference.

[0081] Although the present invention has been described in terms of a certain embodiment, other embodiments apparent to those of ordinary skill in the art also are within the scope of this invention. Thus, various changes and modifications may be made without departing from the spirit and scope of the invention. The scope of the present invention is intended to be defined only by the claims that follow.

What is claimed is:

1. A snowmobile comprising a frame assembly, and a drive assembly depending from the frame assembly, the drive assembly being adapted to abut terrain so as to propel the frame assembly along the terrain, the frame assembly comprising a body frame, and a rear frame portion coupled with the body frame, the rear frame defining a generally horizontally extending bottom section, side sections extending generally vertically from side portions of the bottom section, a rear section extending generally vertically from a rear portion of the bottom section, the bottom, side and rear sections being monolithically formed with each other.

2. The snowmobile as set forth in claim 1, wherein the bottom section forms a recessed space surrounded by the side and rear sections.

3. The snowmobile as set forth in claim 2, wherein a forward end of at least one of the side sections is wider than another portion of the side section.

4. The snowmobile as set forth in claim 2 additionally comprising an internal combustion engine configured to power the drive assembly, and an exhaust system configured to discharge combustion byproducts from the engine, the exhaust system comprising an exhaust silencer defining a chamber therein to reduce exhaust noise, the exhaust silencer at least in part being disposed at the recessed space of the bottom section.

5. The snowmobile as set forth in claim 4, wherein the rear frame defines an opening at the bottom section, the drive assembly comprises a drive belt disposed below the opening.

6. The snowmobile as set forth in claim 1 additionally comprising a grab bar affixed to the rear frame.

7. The snowmobile as set forth in claim 6, wherein the grab bar is affixed to the side sections.

8. The snowmobile as set forth in claim 1, wherein the body frame extends generally horizontally, a rear end of the body frame extends generally upwardly and rearwardly, the rear frame being coupled with the rear end of the body frame.

9. The snowmobile as set forth in claim 1, wherein each one of the side and rear sections forms an inner wall, an outer wall and a top wall connecting the inner and outer walls, the inner and outer walls being spaced apart from each other.

10. The snowmobile as set forth in claim 9, wherein a forward portion of the inner wall is spaced apart from a forward portion of the outer wall greater than another portion of the inner wall.

11. The snowmobile as set forth in claim 1 additionally comprising an internal combustion engine that powers the drive assembly, and a cooling system configured to cool the engine, the cooling system comprising a coolant conduit extending out of the engine, at least one of the side and rear sections forming an inner wall, an outer wall and a top wall connecting the inner and outer walls, the inner and outer walls are spaced apart from each other, the coolant conduit at least in part extending in a space formed between the inner and outer walls.

12. The snowmobile as set forth in claim 11, wherein the body frame forms a foot step, the cooling system comprises a heat exchange device disposed at the foot step.

13. The snowmobile as set forth in claim 1, wherein at least one of the side sections is formed thicker in width at a forward end than another portion of the side section.

14. The snowmobile as set forth in claim 1 additionally comprising an internal combustion engine that powers the drive assembly, an exhaust system configured to discharge combustion byproducts of the engine, the exhaust system comprising an exhaust silencer defining a chamber therein to reduce exhaust noise, the exhaust silencer at least in part being disposed on the bottom section.

15. The snowmobile as set forth in claim 1, wherein the rear frame defines an opening at the bottom section, the drive assembly comprises a drive belt disposed below the opening.

16. A snowmobile comprising a frame assembly, and a drive assembly depending from the frame assembly and adapted to abut terrain so as to propel the frame assembly along the terrain, the frame assembly comprising a body frame, and a rear frame coupled with the body frame, the rear frame defining a generally horizontally extending bottom section, side sections extending generally vertically from side portions of the bottom section, a rear section extending generally vertically from a rear portion of the bottom section, each one of the side and rear sections forming an inner wall, an outer wall and a top wall connecting the inner and outer walls, the inner and outer walls being spaced apart from each other.

17. The snowmobile as set forth in claim 16, wherein a forward portion of the inner wall is spaced apart from a forward portion of the outer wall greater than another portion of the inner wall.

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